Resonating modes of vibrating microcantilevers in liquid

Murali Krishna Ghatkesar, Thomas Braun, Viola Barwich, Jean-Pierre Ramseyer, Christoph Gerber, Martin Hegner, and Hans Peter Lang

National Center of Competence for Research in Nanoscience, Institute of Physics, University of Basel, 4056 Basel, Switzerland

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High sensitivity, portability, multiple target sensing, diverse applicability, and low cost are the driving factors for the development of microcantilever array sensors. This versatile tool has already found application in the field of genomics, proteomics, food engineering, and chemistry. Surface stress (static mode) and mass change (dynamic mode) are the important parameters of interest for microcantilever applied as a sensor. The focus of this article is on the dynamic mode. It is a challenging task to vibrate a microcantilever array in liquid environment. The operation requires efficient coupling of external excitation energy to the microcantilevers avoiding anomalous additional acoustic frequencies originating from the liquid chamber. It also needs a bubble free fluid flow system, optimally focussed parallel laser beams, and sufficient laser power to pass through the air-liquid-air interface for optical detection of cantilever motion. We have operated a microcantilever array in liquid by overcoming these difficulties. A clear and well resolved frequency spectrum with 16 resonant modes of vibration with their corresponding quality factors was obtained. We focused on the comparison of the experimentally obtained eigenfrequencies with those obtained from analytical expressions of different theoretical models.

An array of eight microcantilevers, each 500 μm long, 100 μm wide and 1 μm thick, externally excited by a piezoelement beneath the chip base, was used for the measurements [Fig. 1(a)]. The laser beam deflection detection technique is adapted to monitor resonance frequencies of the eight individual microcantilevers placed in a tiny chamber volume of 6 μl. Liquid was continuously flown at a rate of 10 μl/min. A spectrum of the resonance peaks with corresponding phase information spanning 16 flexural modes until 1 MHz is shown in Fig. 1(b). The phase values are wrapped between 180° and −180°. A phase change of 180° across every peak with a steep slope around the peak position is clearly observed (eigenfrequency).

The experimental eigenfrequencies extracted from the phase spectra [Fig. 1(b)] are shown in Fig. 2(a) as red circles. The corresponding quality factors determined from the amplitude peaks are displayed in Fig. 3(a). Note that the eigenfrequencies deviated by less than 1% among eight cantilevers in an array indicating their similar mechanical characteristics (see supplemental material). The quality factor increase from 1 for mode 1 to 30 for mode 16 indicates that the damping effect from surrounding medium decreases with mode number. The difference between the two adjacent modes in the dynamic mode.
eigenfrequencies increased, while the quality factors were linearly spaced. Note that due to the low quality factor at mode 1 the relative error for the eigenfrequency determination is high.

For a microcantilever vibrating in fluid, there is a strong coupling between the fluid and the cantilever resulting in an inertial loading of the beam (also called virtual mass) which cantilevers have to displace owing to the density and viscosity of the surrounding medium. Therefore, the resonance frequency values in fluid are lower compared to those in vacuum. Furthermore, there is strong damping of the vibration in liquid leading to low quality factors. The motion of the fluid is complex and the modal analysis is difficult. Computational fluid dynamic models with sophisticated numerical techniques considering a three-dimensional (3D) fluid motion around the cantilever have been reported.10–12 In our study, we used analytical expressions from different models which can be readily implemented and compared with experimental data. Models considered are the inviscid model from Chu,13 the Elmer–Dreier model14 for which analytical formulas were derived elsewhere,15 Sader’s viscous model,16 and Sader’s extended viscous model for an arbitrary mode number which accounts for the 3D nature of the flow field around the cantilever beam.17 The general transcendental expression given by all the models for the eigenfrequency of a vibrating cantilever at an arbitrary mode number n in a medium accounting for added apparent mass is

$$f_{R,n} = \frac{c_n^2}{2\pi} \sqrt{\frac{EI}{I_m}} \left[ 1 + \frac{\pi \rho h b^2}{4 \eta} \Gamma(f) \right]$$

where E is the elastic modulus of the material, ρ is the density of the liquid, ρc is the density of the cantilever, l its length, b its width, h its thickness, mc its mass, and I = bh²/12 is its moment of inertia. The constants $\alpha_1 = 1.875$, $\alpha_2 = 4.694$, $\alpha_3 = 7.854$, $\alpha_4 = 11.0$, $\alpha_5 = \pi(n-0.5)$ are solutions of the 1 + cos $\alpha_n \cos a h_n = 0$,18 $\Gamma(f)$ is the normalized hydrodynamic load termed as “hydrodynamic function.” It is a dimensionless function. Different models considered different approaches for the $\Gamma$ function. Equation (1) corresponds to the frequency where the response of the cantilever is shifted by 90° compared to the driving force (eigenfrequency obtained from phase). Note that the frequency corresponding to the amplitude peak (peakfrequency) is shifted toward lower value compared to eigenfrequency due to the damping effect.

The thickness of the cantilevers measured by scanning electron microscopy was 0.817 ± 0.02 μm. The spring constant value calculated from eigenfrequency of 4.5 kHz in air was k = 0.018 N/m. The Reynolds numbers determined by $R_e = 2 \pi \rho f b^2 / 4 \eta$ (Ref. 16) at various resonance modes ranged between 14 and 14570. The density and viscosity values of nanopure water used were 997.8 kg/m³ and 9.772 × 10⁻⁴ kg/m² s⁻¹, respectively.

The predicted eigenfrequencies by the theoretical models (black) are shown in Fig. 2(a) besides the experimental data (red). For more detailed comparison, the relative deviation of frequency values obtained from all the models compared to experimentally obtained eigenfrequencies is shown in Fig. 2(b). Note that for mode 1 all model predictions are within the measurement error of the eigenfrequency. Among all the models, the Elmer–Dreier model matched best but also the extended viscous model in good agreement. The difference between these two models is within the error of the thickness determination of the cantilever (except for mode 1). The good agreement of the Elmer–Dreier model, which is designed for higher modes, is also due to the usage of wide cantilevers reaching inviscid Reynolds numbers at relatively low frequencies. This is in agreement with the validity of the theory for mode number $\geq 4$ above which the effect of viscosity was neglected14 and was also predicted by Van Eysden and Sader.15 The large deviation in the estimation of frequency by viscous model is due to its validity only for low Reynolds number of the order $O(1)$. Moreover it is derived only for fundamental mode.19 The inviscid model deviated more than 30% and underestimated the eigenfrequencies.
considers a constant added apparent mass \( [\Gamma(f) = 1] \) for all modes, which was proved to be not the case for microcantilevers by other models. The extended viscous model derived by Van Eysden and Sader\(^1\) also predicts the frequencies correctly at higher modes. The fundamental frequency matches with the viscous model and for higher modes it follows the Elmer–Dreier model. This indicates that for the dimensions of the cantilever used in the present study, as it resonates from mode 1 to mode 16 the hydrodynamic conditions change from viscous range to inviscid range. Hence the extended viscous model predicts resonance frequencies both at the lower and higher Reynolds number ranges.

We also compared the predicted quality factors with the experimentally obtained values (Fig. 3). An analytical expression is available for the viscous and the extended viscous models to calculate the quality factors but not for the Elmer–Dreier model. The calculated values by viscous model matched quite well with measured data as predicted previously.\(^1\) We used the Elmer–Dreier model to calculate the added apparent mass on the cantilever and its value compared to the cantilever mass (93 ng) is shown in Fig. 3(a). The added apparent mass becomes asymptotically smaller for higher modes. A cantilever has to displace almost 40 times of its mass at fundamental and about 10 times at mode 16. This was predicted in the models by Elmer–Dreier\(^1\) as well as by Van Eysden and Sader.\(^1\) We provide here an experimental verification. These mass ratios are also in agreement with the calibration method for the virtual mass published recently (data not shown).\(^9\)

In conclusion, we present here a well resolved resonance spectrum for cantilevers vibrating in water. The resonance frequency values of 16 flexural modes and their corresponding quality factors of up to 1 MHz are measured. We found that the eigenfrequencies estimated by the Elmer–Dreier model fits well at mode 8 and beyond as well as those predicted by Sader’s extended viscous model. This experimentally proves that the liquid flow around the cantilever truly becomes inviscid at high frequencies. We like to point out that measuring at higher modes has several benefits. First, the quality factors are much higher (a factor of 30 in the present case) leading to smaller experimental error. Second, the virtual mass changes are smaller and change only slightly at higher modes which improves measurement quality and facilitates the calibration of the system. Third, mass sensitivity is at least a couple of orders better at higher mode frequencies\(^20\) than at fundamental mode. With the knowledge of the correct mode number, the virtual mass can be clearly discriminated in mass measurement experiments. The results and the validation of theoretical models will help to design and improve data analysis for biological mass adorption experiments in liquid.\(^9\)

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